Effect of Additional Air Supply for Combustion of Straight Vegetable Oil Blends on Performance of Mini Hot Water Generator

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1. Introduction

The increasing industrialization and motorization of the world has led to a steep rise for the demand of petroleum-based fuels. Petroleum-based fuels are obtained from finite reserves and these limited resources are massively concentrated in certain regions of the world. Therefore, those countries not owning these fuel assets are facing energy/foreign exchange crisis, mainly due to the import of crude petroleum. It has become necessary to look for alternate fuels which can be produced from resources existing nearby regions. The growing energy needs in developing countries, especially in rural areas, give more and more importance to the use of alternate fuels in general and to vegetable oils in particular. This trend has been accelerated by the declining petroleum resources and the role that bio fuels could contribute to reduction of greenhouse gases emission.

Many studies on the characteristics of vegetable oils or their derivatives and their use in diesel engines have been made over the last four decades. The use of vegetable oil in oil burners has been studied by many researchers for the last two decades. Vegetable oils can be used as alternative fuels because they are biodegradable, non-toxic, and clean fuels. Vegetable oils and their derivatives substantially reduces sulfur, carbon monoxide, polycyclic aromatic hydrocarbons, smoke and particulate emissions. Vegetable oils like karanja oil, rapeseed oil, rice bran oil, cottonseed oil, sunflower oil and jatropha oil are being tested by many researchers in diesel engines.

The possibility of using the vegetable oil in raw form and blended form with diesel or biodiesel is being explored in gas turbine and boiler. The researchers have focussed on the selection of fuel properties, techniques for improving the properties of raw fuel, injection methods, mechanism, pattern, injector design, combustion chamber modification, additives, secondary air and optimum fuel air mixture, emissions etc.

Raslavičius et al. [1] studied about the possibilities of biofuel utilization in the transport sector of Lithuania. The CO_2 emission is increasing in Lithuania and this rise could be delivered from all transport subsectors. The emissions of greenhouse gases, air and soil pollution. Raslavicius et al. [2] discussed about the formative breakthrough attitude towards a possible path of Lithuania's sustainable development by utilizing the rape seed oil and its biodiesel for combined heat and power applications.

Zhou et al. [3] interested in biofuel studied and explained increased interest in biofuels globally due to energy security and climate change. According to the IEA report, in 2006 biofuels paid 1% of the world's transport fuel consumption, but in 2030 this could rise to 4%. The production of biodiesel increased by 295%. Phan et al. [4] projected that bioethanol/ biodiesel might replace nearly 10% of diesel fuel consumption within Europe and at Southeast Asia's 5% of total fuel crisis.

Jayed et al. [5] elucidated Thailand's and Indonesia are mainly depending on molasses and cassava and sugarcane for vegetable oil and bio-ethanol production. Whereas Malaysia's making is completely absorbed only in palm diesel, this makes it more susceptible to petroleum and palm oil costs. Still, palm oil's resourceful applications have made it most wanted after vegetable oil in the world. A consequent growth of biofuel plants has leaded a sustainable energy security through biodiesel.

Sanz Requena et al. [6] and Demirbas el al [7] explained the benefits include greenhouse gas reductions, biodegradability, diversification of fuel, sustainability and an additional market for agricultural products. Sustainability should gradually confirm that only advanced biofuels are available for consumers. Sustainability of edible oil can be achieved by mass production of vegetable oil by 20-25% of additional cultivation energy production. The waste or mix oil collected during the extraction of oil from the sunflower oil seed and rice bran oil seed will also improve the sustainability of energy generation from straight vegetable oil. The worthy energy balance joint with a good CO_2 balance endorses vegetable oils as ecologicaleconomical fuels for energy production.

Alonso et al. [8] studied the properties of sunflower, rape seed and soya oil. Properties and chemical compositions of vegetable oil with various proportions with diesel were compared with that of diesel. Singh et al. [9], Yathish et al. [10] and Maran et al. [11] prepared biodiesel of vegetable oil by transesterification process with the aid of methanol and NaOH/KOH/H₂SO₄. Properties of Jatropha and custard apple biodiesel were compared with the properties of diesel. Calorific value of vegetable oil and its biodiesel is lower compared to diesel, but biodiesel has relatively higher viscosity, flash and pour points than that of diesel.

Esteban et al. [12] and Kang et al. [13] studied the effect of preheating on properties of different vegetable oils (VOs) including biodiesel of Rape Seed (RJ), Crude Jatropha (CJ), Sun Flower (SF), Soy Bean (SB), palm, corn and grape seed oils. Research showed that VO is to be preheated to a temperature between 90°C to 120°C minimally to obtain the properties of biodiesel very close to diesel. The authors also investigated the combustion performance and emission characteristics by adjusting the temperature of VO.

Glaude et al. [14] and Chong et al. [15] investigated the combustion of rapeseed Methyl Ester (RME) and Jet-A1 fuel in 6kW gas turbine, where the main air was preheated to the temperature of 350°C at atmospheric pressure. Jet-A1 fuel droplet characteristics such as droplet size, droplet number, density and volume flux were investigated. Czisch et al. [16] developed a rotary atomizer combined with the external mixing gas atomizer. The combined atomizer setup reduces the droplet size up to 210 microns at constant air-fuel ratio. Fan et al. [17] examined a spray of Crude Jatropha oil (CJO), Jatropha methyl ester (JME) and diesel in an air-assist pressure-swirl atomizer. The results showed that atomizer performance depends on the quality of liquid cone, which affects the physical and flow properties.

Raslavicius et al. [18] compared the first generation biofuels with diesel oil and produced power of 340 kW from rapeseed oil methyl ester, which produces the great energy performances and lower emission than that of conventional diesel. Vaitillingom et al. [19] conducted experiments in a 260 kW boiler for different spray nozzles. The properties of vegetable oil and domestic fuel oil were investigated by adjusting the position of the flame stabilizer, opening of air valve, temperature at tip of spray nozzle (80°C to 150°C) and oil injection pressure. It was also noted that there was no ignition under 80°C and 2MPa. Yao et al. [20] studied the influence of viscosity on spray parameters, by mixing of water and glycerol with various volume ratios. The structure of spray was correlated with the viscosity of fuel. Alonso et al. [21] investigated the use of rapeseed oil and diesel mixtures. The viscous effect of the fuel restricted the percentage of rape seed oil in the mixture below 30%. Increase of injection pressure, causes increase in CO emission and decrease in emission of SO₂ and CO₂.

Bazooyar et al. [22] made combustion of rape seed, corn, sunflower, soy bean, olive and rice bran oils in a nonpressurized, water cooled combustion chamber, and estimated the performance and gas emissions. Thermal efficiency was lower in the range of 12% to 21% at a low fuel pressure of 8.274 bar, and 21% to 26% at higher fuel pressure of 19.305bar, for different fuel blends.

Ghorbani et al. [23] compared the combustion efficiency and flue gas emission for biodiesels of soy bean oil and sun flower oil at two energy levels i.e., 219 kJ/hr and 249 kJ/hr. Combustion efficiency decreases as air flow is increased. Soy bean bio diesel was more efficient than sunflower biodiesel at higher energy rate. 20% blend of Soy bean biodiesel produces little lower combustion efficiency than that of diesel but CO_2 and SO_2 emission reduces by 4.2% and 7% respectively.

Demirbas [1] tested particulate matter in CI engine it was found that the PM level for 20% biodiesel reduced about 8.9% and 100% biodiesel reduced about 55.3%. Alonso et al. [8] found that the particulate matter in vegetable oil is reduced to 65% than petroleum diesel. Dizayi et al. [24] identified about the PM concern for vegetable oil it lies between PM 10 to PM 2.5, (i.e. diameter of micro meters).PM emissions for vegetable oil blends were lower than diesel up to 65-75%.

Daho et al. [25] studied the influence of combustion parameters such as equivalence ratio and fuel pressure on combustion and emission performance of blends of 10 to 70% refined cottonseed oil (RCO) with domestic fuel-oil (DFO) in a non-modified DFO burner.

From the available literatures, it is clear that a very few experimental studies have been reported for the investigation on combustion of straight vegetable oils and blends in burners used in furnaces of boilers. Further, limited attempts

2. Experimental setup

emissions.

The schematic diagram of the experimental setup used in the present study is shown in Fig. 1. The setup has been designed and developed to conduct experiments to study the combustion performance and emission trend by using straight vegetable oil and its blends with diesel. The set up consists of a commercial oil burner, hot water generator and an additional air supply system.

thermal efficiency of hot water generator and lowering the

The mini hot water generator is a fire tube boiler having 35 kg water capacity. It has four fire tubes of diameter 38 mm, passing the hot gases from the furnace to the smoke tube heating the water in the boiler drum. The main drum and furnace have been insulated by using glass wool and asbestos rope and covered with heat resistance sheet. The commercial oil burner is of swirl type burner with a capacity of 150 g/min, which atomizes the liquid oil into tiny particles using forced air from the centrifugal blower. The main air is supplied by a blower having the capacity of $0.0472m^3/s$.



Fig. 1 Experimental setup of hot water generator

The volume flow rates of main and additional air are measured by using orifice meters and the mass flow rate is calculated using local pressure and temperature. The fuel is supplied from the overhead fuel tank, and the mass flow rate of fuel is measured with the help of a digital weighing machine. The thermocouples are fixed to measure the temperatures of water and flue gases. Thermocouple sensors were located from the top of the shell so that the sensor end is always dipped into water in the boiler shell and not touching the tube or side wall.

Thermocouples used are of Chromel/Alumel K-type, (0-1200°C) with the accuracy of \pm 0.5% of measured value and resolution of \pm 0.1°C.A CRYPTON 290-EN2 type gas analyser has been used to measure the exhaust emissions from the hot water generator setup. The analyser has an accuracy of \pm 5% of measured values of HC, NO_x, CO and CO₂ with a resolution of 1 ppm for HC and NO_x and of 0.1 % for CO and CO₂. For O₂ accuracy is \pm 0.2 % of measured value and resolution of 0.1 %. The range of gas analyser has been presented in Table 1.

3. Experiments and results

Experiments have been conducted to investigate the

combustion performance and emission levels during the combustion of straight vegetable oils blended with diesel in a hot water generator with additional (WA) air supply and without additional (WOA) air supply. The details of fuel blends used in the present study are presented in Table 2.

Each fuel blend is named with the name of the vegetable oil and its percentage in the blend. For example, 25 % sunflower oil in the blend is named as SFVO 25 and 50 % rice bran oil in the blend is named as RBVO 50. For each case, thermal efficiency and emission levels have been found and compared in the following sections. Calorific values of the respective SVO blends were found by using the bomb calorimeter and are presented in Table 2.

Specifications of gas analyser

Measuring Gas	Maximum Range
СО	0-10 (%)
HC	0-20000 (ppm)
CO ₂	0-20 (%)
O2	0-25 (%)
NO _x	0-1000 (ppm)

Table 2

Fuel blends

Fuel	Diesel Proportion (By Volume)	SVO Proportion (By Volume)	Fuel Blend	Higher Calorific value kJ/kg K
Diesel + Sunflower oil	Diesel 100%	Sunflower oil 0%	Pure Diesel	42414
	Diesel 75%	Sunflower oil 25%	SFVO 25	40702
	Diesel 50%	Sunflower oil 50%	SFVO 50	38975
Diesel + Rice Bran oil	Diesel 100%	Rice Bran Oil 0%	Pure Diesel	42414
	Diesel 75%	Rice Bran Oil 25%	RBVO 25	40118
	Diesel 50%	Rice Bran Oil 50%	RBVO 50	37810

Table 3

Results for combustion of SVO blends with diesel in a commercial oil burner with main air supply

S1		Mass of the Fuel	Temperature (°C)		Rise in	Host Input	Host Output	Efficiency
DI.	Fuel	Consumed (Ise)	Initial	Final	Temperature			(m) 0/
NO		Consumed (kg)	(T_i)	(T_f)	(°C)	(Q_i) KJ	(\mathcal{Q}_o) KJ	(1) %
1	Diesel	0.295	40	80	40	12512.1	4689	37.48
2	SFVO 25	0.440	40	80	40	17908.9	4689	26.18
3	SFVO 50	0.560	40	80	40	21826.0	4689	21.49
4	RBVO 25	0.408	40	80	40	16368.1	4689	28.65
5	RBVO 50	0.528	40	80	40	19963.7	4689	23.49

3.1. Results of combustion of SVO blends with main air supply

28 kg of water (m_w) in the mini hot water generator shell is heated by firing different fuels as listed in Table 2, so that the temperature of water is increased from 40°C (T_i) to 80°C (T_f) . Mass of the fuel in the fuel (m_f) tank is weighed at the beginning and end of the heating process recorded as m_i and m_f respectively.

The heat output (Q_o) during the heating of water is calculated from the mass of water and temperature rise in water i.e., $Q_o = m_{w.} C_p (T_f - T_i)$, kJ and the heat input is by fuel consumption and calorific value i.e., $(Q_i) = m_{f} CV$, kJ. Finally, the efficiency (η) of mini hot water generator is calculated as the ratio between Q_o and Q_i .

The measurement and calculated results for the case of burning of fuels fired with main air supply alone using the commercial oil burner have been presented in Table 3.

Fig. 2 depicts the comparison of thermal efficiency of SVO blends fired by a commercial oil burner with main air supply. The thermal efficiency is in general, maximum for diesel and it is 37.48%. RBVO 25 and SFVO 25 blends have thermal efficiency of 28.65% and 26.18% respectively. RBVO 50 and SFV 50 blends have thermal efficiency of 23.49% and 21.49% respectively.

The thermal efficiency of the system is found to be low for all the blends of straight vegetable oils compared to the case of diesel. This may be due to the insufficient amount of air supplied, poor atomization of the fuel and high viscosity of the oil. Hence, a further study has been carried out to find the influence of additional air supply to the burner to improve the efficiency of the system when fired by SVO blends.



Fig. 2 Thermal efficiency of SVO blends fired by a commercial oil burner with main air supply

3.2. Results of combustion of SVO blends with main and additional air supply

The combustion of blends of RBVO and SFVO oils has been carried out with main and additional air supply (40% and 80% additional air). The arrangement of additional air supply system is shown in Fig. 3.

The measurement and calculated results for the case of burning of fuels fired with main air and 80% additional air supply using the commercial oil burner have been presented in Table 4.

Fig. 4 presents the comparison of thermal efficiency of the system when fired with various SVO blends using main and additional air supply. The use of additional air supply increases the thermal efficiency; by improving the mixing of fuel and air molecules. It is clear that the improvement in thermal efficiency of the combustion system is more for SVO blends than that of diesel when the extra air is supplied. This may be due to the requirement of sufficient oxygen molecules for complete combustion in case of high viscous SVO blends.

The thermal efficiency of diesel increases from 37.48% to 39.48% and the percentage improvement in the efficiency is only 5.34%. RBVO 25 and SFVO 25 blends have thermal efficiency of 33.98% and 31.31% and the corresponding percentage increase in the efficiency is 18.6% and 19.59% respectively. Similarly, RBVO 50 and SFVO 50 blends have thermal efficiency of 30.7% and 25.93% and the corresponding percentage increase in the efficiency is 30.69% and 20.66% respectively.



Fig. 3 Additional air supply system

Table 4

Results for combustion of SVO blends with main and 80 % additional air supply

		Mass of the Fuel	Temperature (°C)		Rise in	Heat Input	Heat Output	Efficiency
Sl.No	Fuel	Consumed	Initial	Final	Temperature	(0) kI	(O) kI	efficiency
		(kg)	(T_i)	(T_f)	(°C)	(Q_i) KJ	(\mathcal{Q}_0) KJ	(1) %
1	Diesel	0.280	40	80	40	11875.9	4689	39.48
2	SFVO 25	0.368	40	80	40	14978.3	4689	31.31
3	SFVO 50	0.464	40	80	40	18084.4	4689	25.93
4	RBVO 25	0.344	40	80	40	13800.5	4689	33.98
5	RBVO 50	0.404	40	80	40	15275.2	4689	30.70



Fig. 4 Comparison of thermal efficiency of SVO blends fired by a commercial oil burner with and without additional air supply

The extra oxygen supplied in the form of additional air supply helps in providing better mixing with the fuel and improving the combustion. Especially the efficiency of SVO blend with higher percentage improves more than that of lower percentage in the blend. The unburned fuel molecules in the combustion products due to incomplete combustion get combusted with the additional air supply. As more numbers of unburned molecules are present in the combustion products of higher percent percentage SVO (50%), the improvement in the efficiency is also more.

4. Emissions

4.1. Carbon monoxide emission

Fig. 5 shows the levels of carbon monoxide emission in the exhaust gases from the mini hot water generator fired with diesel, SFVO 25, SFVO 50, RBVO 25 and RBVO 50.The measured values for CO emission during the combustion of fuels using main air alone and with the use 80% of additional air supply have been compared.



Fig. 5 Carbon monoxide emission for various fuel blends

CO emission significantly increases with the addition of SVO in the fuel blend at higher percentage when the combustion is carried out without additional air supply. This may be due to incomplete combustion of viscous vegetable oil, insufficient O_2 and poor mixing of oxygen and oil. The CO emission is slightly higher for rice bran oil blend than that of sunflower oil.

The results for additional air supply (80%) show that the CO emission is slightly lower for rice bran oil blend than that of sunflower oil. It may be due to the better mixing of air and fuel droplet and combustion of rice bran oil with the supply of additional air. It also reduces the level of CO emission by about 40 % in all the cases. This higher percentage may be due to the conversion of carbon mono-oxide molecule into carbon di oxide and the dilution of concentriton due to volume of additional air in the combustion products.

4.2. Carbon dioxide emission

 CO_2 emission is less than that of diesel for all the fuel blends of sunflower and ricebran oil. Further, the percentage of CO_2 emission is found to decrease for all the blends, when the additional air supply is introduced. But this drop in CO_2 is due to conversion of unburnt fuel molecules as well as excess air supplied in the form of additional air to the combustion system. This is due to dilution of gas mixure, where the concentration of CO_2 decreases with excess air.



Fig. 6 Carbon dioxide emission for various fuel blends

4.3. Nitrogen oxides emission

The of nitrogen oxide emissions have been measured for burning of different blends of SFVO and RBVO with diesel, fired with main and additional air supply and the results are shown in Fig. 7. The NO_x emission is higher for diesel than SVO blends. Figure shows the influence of blend on the nitrogen oxides emission.

The emission of nitrogen oxides is higher for diesel for both the cases, with and without the supply of additional air and the levels have been recorded as 52 ppm and 48 ppm respectively. For SFVO 25, the NO_x emission is slightly lower than that of diesel and it is 38 ppm and 39 ppm for the cases, with and without additional air supply respectively. Further higher percentage of oil in the 50 % blend decreases the NOx emission up to 24 and 22 ppm. NO_x emission for rice bran oil is quite higher than that of sunflower oil blends.



Fig. 7 Nitrogen oxides emission for various fuel blends

The NO_x emission of RBVO 25 blend is 45 ppm and 42 ppm for WOA and WA respectively. The emission further decreases to 34 ppm for the case, without additional air and 31 ppm for the case, with additional air, for RBVO 50. The lower in NO_x for the case of additional air supply is due to more amount of nitrogen and oxygen present in the excess air and lower furnace temperature.

5. Conclusions

In the present work, an experimental study has been carried out to improve the performance of mini hot water generator using additional air supply. The following are the conclusions from the present study. The additional air supply with the main air supply enhances the combustion performance, leading to increase in thermal efficiency and decrease in CO emission.

1. The efficiency of mini hot water generator fired with diesel increases from 37.48% to 39.48% when 80 % additional air is supplied.

2. The efficiency of mini hot water generator fired with SFVO 25 and RBVO 25 increases by 19.59% and 18.6% respectively when the burner is supplied with 80% additional air supply and it is 20.66% and 30.69% rise for SFVO 50 and RBVO 50 respectively, fired with additional air supply.

3. The CO emission is almost in the same range that of diesel. But CO_2 decreases by 34.6 % and 33.4 % for 25% blends of straight sunflower oil and rice bran oil blend when additional air is supplied. For 50% blends of straight sunflower oil and rice bran oil blend, the CO_2 emission decreases by 37.9% and 35.4% respectively. The level of NO_x emission decreases for SVO blended with diesel and it is in the range of 22 to 52 ppm.

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EFFECT OF ADDITIONAL AIR SUPPLY FOR COMBUSTION OF STRAIGHT VEGETABLE OIL BLENDS ON PERFORMANCE OF MINI HOT WATER GENERATOR

Summary

This paper presents the results of an experimental investigation on the efficient combustion of blends of Straight Vegetable Oil (SVO) with diesel, fired in a non-commercial hot water generator. The high viscosity and poor atomization characteristics of straight vegetable oil lower the combustion and thermal efficiency of the system. An additional air supply system has been used to provide sufficient amount of air into the furnace to improve the mixing of fuel droplets and oxygen in the air. Thermal efficiency and emission characteristics of 25% and 50% blends of straight sunflower oil and rice bran oil with diesel have been studied with and without additional air supply. Thermal efficiency of the hot water generator has been found to be increased when the quantity of additional air supply is increased. Moreover, the emissions of CO and HC have been reduced by the additional air supply.

Keywords: straight vegetable oil, blend, diesel, additional air supply, combustion, emission.

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